

IN THE SPECIFICATION

Please correct the paragraph that begins on page 4, at line 29, to read as follows:

Water passing through the coil 20 in the heat exchanger 18 is pumped through a closed circuit by a pump 22, and the pump 22 receives water from a water cooling tower 24, which has a cooling water supply 24-1 and a cooling water return 24-2, either from an inlet 24a to the cooling water supply 24-1 of the water tower 24 (the broken line flow path), where the water temperature will be approximately 85° F, or from an outlet 24b from the cooling water return 24-2 (the solid line flow path) where the water will be approximately 100° F, or partly from each.

In that regard, a temperature controlled 3-way diverter or mixing valve 26 is provided to mix water from the inlet 24a with water from the outlet 24b to introduce water at a controlled temperature from the pump 22 to the heat exchanger 18, and a pressure regulator 28 is provided in the line from the inlet 24a to equalize the pressure therein with pressure in the lines from the outlet 24b. Water from the heat exchanger 18 is then returned to the water cooling tower 24 through a return line 48 to be mixed with water entering the water cooling tower 24, and various pressure indicators TI and pressure indicators PI are placed in the water circuit that passes water through the heat exchanger 18 to ensure proper

temperature and pressure conditions exist therein. The water circulating through the heat exchanger 18 has ball valves 30, 32 at the water inlet thereto and the water outlet therefrom, respectively, a ball valve 34 in a drain line from the heat exchanger 18 and a ball valve 36 in a vent line from the heat exchanger 18. The water circulating through the heat exchanger 18 has a check valve 38 at a location upstream of the heat exchanger 18 to prevent reverse flow to the heat exchanger 18, and a strainer 40 immediately upstream of the heat exchanger 18 to prevent any particles in the water flowing through the system from clogging the coil 20. Ball valves 42, 44, 46 are also provided at the outlet from the cooling water supply 24-2 of the water cooling tower 24, the inlet to the cooling water return 24-2 of the cooling water tower 24 and at the outlet from the cooling water return 24-1 of the cooling water tower 24, respectively.

Please amend the paragraph that begins on page 6, at line 25, to read as follows:

When it is desired to cool the cooling air flowing to the manifold 112, a second indirect heat exchanger 160 is provided at a location downstream of the blower 114. The heat exchanger 160 has a sinuous coil 162 therein, and water at an approximate temperature of 85° F. is permitted to flow through the coil 162 to cool air from the blower 114 before it passes to the manifold 112. In that regard, the air from the

blower 114 will be somewhat hotter than air from the air plenum 116 that enters the blower 114 due to the heat imparted to the air by its compression in the blower 114 and frictional losses in the blower 114. In any case, when the desired temperature of the cooling air in the manifold 112 is 100° F., this can readily be achieved by cooling the air in the second heat exchanger 160 with a cooling water supply of 85° F. Such cooling water supply to the second heat exchanger 160 is obtained from an outlet 124a of the cooling water supply 124-1 of the water cooling tower 124, and the flow rate of water to the second heat exchanger 160 is controlled by a temperature control flow control valve 164. In the system of Fig. 2 then, heating of a cooling air supply to maintain it at an acceptable temperature for use, when required, is done by the indirect heat exchanger 118, and cooling of such air supply, to maintain it at an acceptable temperature for use, when required, is done by the second heat exchanger 160. When the operation of the flow control valves 126, 164 is properly coordinated, for example, by a common temperature control algorithm, the temperature of the cooling air in the manifold 112 can be plus/minus 1° F. of the desired set point temperature, typically 100° F., during typical day to night inlet air temperature variations that occur in Northern locations during winter months. In any case, even with manual control of the flow control valves 126, 164, the temperature of the cooling air in the manifold 112 can be held within much narrower limits than [as] has heretofore been achieved with known cooling air temperature control systems.

Please amend the paragraph that begins on page 7, at line 22, to read as follows:

The water circuit through the heat exchanger 118 has ball valves 130, 132 at the water inlet thereto and the water outlet therefrom,

respectively, a ball valve 134 in a drain line from the heat exchanger 118 and a ball valve 136 in a vent line from the heat exchanger 118. The water circuit through the heat exchanger 118 also has a strainer 140 at a location upstream of the heat exchanger 118 to prevent any particles in the water flowing through the system from clogging the coil 120. Ball valves 142, 144, 146 are also provided at the outlet from the cooling water return 124-2 of the water cooling tower 124, at the inlet to the cooling water return 124-2 of the water cooling tower 124, and the inlet 146 from the cooling water supply 124-1 of the water cooling tower 124, respectively. A temperature indicator TI and various pressure indicators PI are also provided in the water line that leads to the coil 120 of the heat exchanger 118. The water circuit through the heat exchanger 118 also has a check valve 138 at a location downstream from the heat exchanger 118 to prevent reverse flow through the heat exchanger 118[.] , and a return line 148 to return spent water to the cooling tower 124.

IN THE DRAWING

Please substitute the enclosed 2 sheets of drawing figures for their originally-filed counterparts.